90% of our production is exported to more than 40 countries, in each country.
You will find a Rollix representative or member of our distribution network at your disposal, to help with all aspects of your requirements as quickly and efficiently as possible.
Our engineering and design department will propose you innovating solutions, using the most modern software, as well as calculation by finite elements.
This catalogue specifies the shortest delivery times for standard slewing rings from our stock in France and abroad.
In order to match closely customer’s requirements Rollix has developed 2 new ranges of products: “RLX COMPACT” for applications requiring high precision and “RLX-BT” for bottling industries.
We design parts specifically adapted to your needs and we own several thousands of other references that we keep at your disposal.
Rollix specialises in preloaded slewing rings. Thanks to our process of grinding of the raceway and to systematic matching, we offer you a slewing ring that is as precise as possible throughout its lifetime.
Thanks to simultaneous engineering, we produce a product that answers the most specific requests of the most demanding customers (slewing rings made of titanium, aluminium etc…) Your most severe requirements (loads, tightness, speed, noise etc…) are validated on our test benches.
Our ISO 9001 and ISO 14000 certification guarantees a product designed and produced according to the rule book.
Our slewing rings conform to standards DNV, Lloyd, GL etc, which are recognized worldwide.
## WORLD-WIDE LOCATIONS

**ROLLIX** is represented in the following countries:

<table>
<thead>
<tr>
<th>Argentina</th>
<th>Greece</th>
<th>New Zealand</th>
</tr>
</thead>
<tbody>
<tr>
<td>Austria</td>
<td>Hong Kong</td>
<td>Norway</td>
</tr>
<tr>
<td>Australia</td>
<td>India</td>
<td>Philippines</td>
</tr>
<tr>
<td>Belgium</td>
<td>Iran</td>
<td>Poland</td>
</tr>
<tr>
<td>Brazil</td>
<td>Ireland</td>
<td>Portugal</td>
</tr>
<tr>
<td>Canada</td>
<td>Israel</td>
<td>Russia</td>
</tr>
<tr>
<td>China</td>
<td>Italy</td>
<td>Slovakia</td>
</tr>
<tr>
<td>Czech Republic</td>
<td>Korea</td>
<td>South Africa</td>
</tr>
<tr>
<td>Denmark</td>
<td>Latvia</td>
<td>Spain</td>
</tr>
<tr>
<td>Egypt</td>
<td>Lithuania</td>
<td>Sweden</td>
</tr>
<tr>
<td>Estonia</td>
<td>Malaysia</td>
<td>Switzerland</td>
</tr>
<tr>
<td>Finland</td>
<td>Mexico</td>
<td>Taiwan</td>
</tr>
<tr>
<td>Germany</td>
<td>Morocco</td>
<td>Thailand</td>
</tr>
<tr>
<td>Great Britain</td>
<td>Netherlands</td>
<td>Turkey</td>
</tr>
<tr>
<td>United States</td>
<td></td>
<td></td>
</tr>
</tbody>
</table>
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BUSINESS UNITS AND PRESENTATION

- WIND TURBINE
- SOLAR
- MEDICAL
- PRECISION
- PUBLIC WORKS
- CRANES
- PACKAGING
- WATER TREATMENT
- OFFSHORE
- MINING
- GREEN INDUSTRY
- HANDLING
- DEFENCE
- TRANSPORT
- INDUSTRY
- CONSTRUCTION
- CAPABILITY
BUSINESS UNIT

WIND TURBINE

50% of the wind turbines in the world use Rollix slewing rings for blade and yaw. Our design office is specialised to design specific bearings with 20 years life time. Our special sealing system gives completely tight bearings and we can propose integrated bearings with automatic lubrication system. Our finishing by grinding is the state of the art for the double row slewing rings. From 5kw up to 5 MW Rollix supplies all the manufacturers in this field.

SOLAR

The rotation of the solar panels is a good solution to increase the production of energy. Because of our very compact design, Rollix supplies the biggest solar farms in Europe.
BUSINESS UNIT

MEDICAL

The Rollix Medical slewing rings are characterised by a high rotating speed, constant rotating torque and low noise under rotation controlled in our anecoid room. The perfect circle and the high precision of our bearing are the result of our specific finishing and the matching of the rings and the rolling elements. We propose also innovative sealing solutions to avoid any leakage in a clean environment.

PRECISION

The success of the Rollix slewing ring is the result of the precision of our finishing by grinding. All our production is perfectly under control and the matching of the rings and the rolling elements, gives batch after batch, the certainty to obtain the right bearing for this very demanding application. We propose also motor torque bearings. Our tailor made design follows perfectly the specific requirements of the customers. The most famous worldwide robots and machine tools manufacturers choose Rollix all around the world.
BUSINESS UNIT

PUBLIC WORKS
The Rollix slewing ring, a very robust mechanical part, is particularly adapted to these applications. Our process control gives the customer a very simple solution to set up the pinion (see page 45) and to save production time. The most famous worldwide manufacturers choose Rollix for our optimised design and the quality of our service.

CRANES
Because of the light structure of the crane, the Rollix slewing ring is an essential and strategic part. Our design matches perfectly the life time required by the customer. The quality and the precision of our manufacturing is the key to ensure good running after many cranes dismantling. In this application the Rollix slewing ring helps construct buildings everyday all around the world with total security.
BUSINESS UNIT

PACKAGING

Large diameters and thin sections are the main characteristics of this application. These particular bearings require strict process control. The Rollix slewing rings are very well-known in the world to accept the highest rotating speed and to control the temperature during working 24 hours a day. Our quality gives a very low level of maintenance. The Rollix slewing rings manufacture millions of bottles all around the world everyday.

WATER TREATMENT

Compared with standard bearings, the Rollix slewing ring is a good solution to reduce the assembling time, to simplify the structure and to increase the capacity of the whole system. The high capacity of our gear is the result of our process control. Many towns around the world use our product every day to treat water. Our worldwide company structure gives strong support to customers and to end users.
BUSINESS UNIT

OFFSHORE

High speeds, high loads and salty environment require strong and precise components. The Rollix slewing rings is the perfect answer to the offshore industry. Our specific design satisfy the main manufacturers of onboard and harbour cranes, winches, davits and rubber tired gantry cranes as well as propellers. We propose innovative solutions of sealing to completely protect the bearings from the environment. Rollix follows the international certification standards.

MINING

Stacker reclaimers, tunnel boring machines and excavators are very demanding applications. The high capacity of the Rollix slewing rings is obtained by the control of the hardening process of the raceways and the gears. Our logistic department supplies our product everywhere in the world even large diameter slewing rings.
BUSINESS UNIT

GREEN INDUSTRY

Forest and de-barker machines require strong components. The most famous worldwide manufacturers choose Rollix for our optimised design and the capacity of our slewing rings to work in a tough environment. Our finishing by grinding brings a valid pre-loading which is very important for the life time of the bearing and the comfort of the forest machine’s drivers. Our slewing rings accept the shocks and the high rotating speed of the de-barkers as well as the temperature of the coldest forests.

HANDLING

The stiffness of the Rollix slewing ring is a perfect solution for this type of application. Our special design gives very compact bearings, which is essential for this product. Our process control gives the customer a very simple solution to set up the pinion (see page 45) in order to save time during the mounting of the machines.
BUSINESS UNIT

DEFENCE

The Rollix slewing ring has a strong reputation for civilian or military applications. We manufacture steel, aluminium and titanium bearings for the most demanding customers in the world. Our precision is obtained by a specific process of grinding. Our design office proposes specific solutions for all types of environments. Capacity, low torque, stiffness and high rotating speed are our main characteristics.

TRANSPORT

The Rollix slewing ring permits the bogie orientation and the carriage articulation. The stress spectrum applied to these slewing rings is very specific. Our know-how answers perfectly the protection from contaminants (salt, sand, rain and mud). We can also integrate special braking devices. The Rollix slewing rings transport millions of people all around the world with train, tram and metro. We also supply slewing rings for the rotation of the truck’s wheels.
VARIOUS INDUSTRY

Rollix supplies slewing rings for a wide range of applications throughout the industry. Our design office will assist you to select the most effective solution price and capacity-wise. Our staff and agents worldwide will process your enquiries and provide full support to meet your customer’s requirements.

The loads and working conditions defined and gathered by the customer in the IT ETR 911 data sheet enables the ROLLIX Engineering Department to select the most suitable product.
CONSTRUCTION

BASIC CONSTRUCTION

- External or internal gear
- Special steel, one-piece rings with or without gear
- Blind or through holes, untapped or tapped
- Specially designed profile seal
- Lubrication holes
- Filler plug
- Raceways
- Balls or rollers made of bearing steel
CONSTRUCTION

MATERIALS

For the slewing ring manufacture, ROLLIX selects the best material specifications to suit most foreseen usages. These materials are manufactured by approved steel works. Inspections are carried out at each significant step of manufacturing what makes it possible to warrant the quality of the products. Most of the time, we select high-grade Carbon steels which permit us to meet all operating requirements. Core hardening and tempering are carried out when the applied stresses dictate these processes.

<table>
<thead>
<tr>
<th>STANDARD GRADES</th>
<th>Normalized state</th>
<th>Core hardened state</th>
</tr>
</thead>
<tbody>
<tr>
<td>Improved XC 45</td>
<td>Code letter Z</td>
<td>Code letter X</td>
</tr>
<tr>
<td>42 Cr Mo 4</td>
<td>code letter N</td>
<td>Code letter D</td>
</tr>
</tbody>
</table>
CONSTRUCTION

MATERIALS

EQUIVALENT STANDARDS IN VARIOUS COUNTRIES:
The following table shows our standard steels with the nearest equivalent foreign grades.

<table>
<thead>
<tr>
<th>COUNTRIES</th>
<th>STANDARD</th>
<th>XC45 Code Z/X</th>
<th>42CrMo4 Code N/D</th>
</tr>
</thead>
<tbody>
<tr>
<td>GERMANY</td>
<td>DIN</td>
<td>Ck 45</td>
<td>42CrMo4</td>
</tr>
<tr>
<td>U.K.</td>
<td>B.S.</td>
<td>080M46</td>
<td>708M40</td>
</tr>
<tr>
<td>U.S.A.</td>
<td>AISI</td>
<td>16845</td>
<td>4142</td>
</tr>
<tr>
<td>ITALY</td>
<td>UNI</td>
<td>C45</td>
<td>42CrMo4</td>
</tr>
<tr>
<td>JAPAN</td>
<td>JIS</td>
<td>S45C</td>
<td>SNB7</td>
</tr>
<tr>
<td>SPAIN</td>
<td>UNE</td>
<td>C45K (F1140)</td>
<td>42CrMo4 (F8232)</td>
</tr>
<tr>
<td>SWEDEN</td>
<td>SSSTAHL</td>
<td>1672</td>
<td>2244</td>
</tr>
<tr>
<td>AUSTRALIA</td>
<td>ASA</td>
<td>AS 1442-1045</td>
<td>AS 1444-4140</td>
</tr>
</tbody>
</table>

OTHERS MATERIALS:
Specific factors of some applications or functional requirements may result in the use of materials such as:
- Stainless steel.
- Structural hardening steels or alloys.
- Special steels for hardening under controlled atmosphere.
- Case or nitried steel.
- Special steels for very low temperature.
- Aluminium-based alloys.
- Titanium alloys.
- Plastic, composite materials.

COMPONENTS
ROLLIX selects rolling elements: balls and rollers with very specific characteristics.
Their assembly requires particular caution.
Any action of dismantling bearings or replacement of rolling elements is definitely inadvisable and cancels the ROLLIX warranty.
CONSTRUCTION

HEAT TREATMENTS

The slewing ring transmits loads from a turning part towards the fixed part of a mechanism. The applied stresses of the rolling elements on the raceways are calculated according to the Hertz laws and the modern criteria of plasticity.

ROLLIX carries out localized hardening treatments which makes it possible to satisfy these criteria just as well in terms of surface pressure as well as subsurface fatigue.

In all cases, induction or flame hardening permits us to achieve the necessary hardness patterns and a sufficient depth simultaneously.

In process systematic inspections during production enable us to warrant the quality and the reliability of the slewing ring treatment, according to ROLLIX specifications.

According to the level of applied stresses and if geometry allows it, the same kind of hardening process may be applied on gear teeth.

Our calculation models allow us to select the kind of treatment required. Other processes of local hardening may also be provided, for example: case hardening, nitriding, etc.
CAPABILITY

TEMPERATURE

The normal working temperature of the slewing rings ranges from -25 up to +70˚ C. Lower or higher temperatures are possible but require special design provisions by our Engineering Department.

SPEED

The slewing rings can work whether by oscillating motion or continuous motion. It is necessary to check that the circumferential raceway speed remains within the acceptable limits of respective bearing’s capacity.

ENVIRONMENT

In case when the working environment is particularly aggressive:

- sea atmosphere,
- dusty or abrasive environment (sand, coal...),

particular protection devices must be incorporated such as:

- labyrinths,
- shields,
- oil-bath.

Preventive maintenance operations will be increased to ensure normal operation.

SHOCKS, VIBRATIONS

If the slewing rings are continuously solicited by shocks and/or vibrations, the customer must mention it in the specifications in order to design the right piece.

<table>
<thead>
<tr>
<th>BEARING TYPE</th>
<th>LUBRICATION TYPE</th>
<th>LIMITED SPEED (n. Dm)</th>
</tr>
</thead>
<tbody>
<tr>
<td>Crossed rollers raceway</td>
<td>Standard grease</td>
<td>24 000 to 35 000</td>
</tr>
<tr>
<td>Balls type raceway</td>
<td>Standard grease</td>
<td>40 000 to 65 000</td>
</tr>
<tr>
<td>Balls type with cage</td>
<td>Grease or oil</td>
<td>70 000 to 130 000</td>
</tr>
<tr>
<td>Specific design</td>
<td>Oil or special greases</td>
<td>Consult us</td>
</tr>
</tbody>
</table>

In any case, it is essential to consult our Engineering Department with the precise operating conditions.
SLEWING RING SELECTION

- SLEWING RINGS FUNCTIONS
- OTHER FEATURES
SLEWING RING FUNCTIONS

LOADS SELECTION

The slewing ring, being the link between a mobile element and a fixed base, must have the capacity to transmit the stresses of the mobile part towards the base. It is necessary to accurately define all the actual stresses applied so that a suitable slewing ring with adequate capacity can be selected. This should include the effects due to masses and inertias of the payloads and structures.

It is necessary to distinguish the fixed loads and the variable loads as well as the effects due to dynamic loads, the latter constituting "fatigue" stresses. The direction of forces affecting the slewing ring must be well defined so that the active tilting moment can be established.

We distinguish:
- The AXIAL LOADS whose direction is parallel to the slewing ring rotation axis. The resultant of these loads is called $F_A$.
- The RADIAL LOADS contained in planes perpendicular to the rotation axis. The resultant of these loads is called $F_R$. 

![AXIAL LOADS](image)

![RADIAL LOADS](image)
SLEWING RING FUNCTIONS

LOADS SELECTION

- **TILTING MOMENTS** (bending): in planes parallel to the rotation axis. The resulting moment working in the plane containing the rotation axis is called $M_T$.

- **SLEWING TORQUE** $C_D$ controls the slewing ring rotation.

**CALCULATION OF THE EQUIVALENT LOAD**

For the calculation, the resultant of the radial loads $F_R$ is transposed into an equivalent axial load using a factor $K_R$ as follows:

For standard slewing ring:

- if $F_R < 0.25$
  - $F_A$
  - $K_R = 0.5$
- if $0.25 < F_R < 1$
  - $F_A$
  - $K_R = 1.5$
- if $F_R > 1$
  - $F_A$
  - $K_R = 2.4$

For light series and solid sections: $K_R = 3.225$

The equivalent loads $F_{eq}$ can be obtained by the following formula:

- For the horizontally mounted slewing rings: vertical rotation axis:
  - $F_{eq} = F_A + K_R \cdot F_R$

- For the vertically mounted slewing rings: horizontal rotation axis:
  - $F_{eq} = F_A + 1.2 \cdot K_R \cdot F_R$

**TILTING MOMENT**

$M_T$

$C_D$
## DESIGN DATA SHEET

### 1. COMPANY
- Tel.: 
- Fax: 

### 2. Person in charge of the project
- Fonction: 

#### 2.1 DESCRIPTION OF THE APPLICATION + sketch (to be attached with loads applied)

#### 2.2 Machine / project reference
- New project: Yes [ ] No [ ]

#### 2.3 Slewing ring position
- Horiz. [ ] Vert. [ ] Other [ ]
- Utilisation mode: Continu. [ ] Intermittent [ ] Other [ ]

### 3. LOADS ON THE BEARING (including structural loads)
- Applied [ ] Suspended [ ]

#### 3.1 Static
- Nominal
- Maximum
- Test

#### 3.2 Dynamic
- Nominal
- Maximum
- Test

#### 3.3 Utilisation
- % Time

#### 3.4 Axial
- kN

#### 3.5 Radial
- kN

#### 3.6 Moment
- kNm

#### 3.7 Rotating ring
- Ext [ ] Int [ ]
- Speed (RPM)

#### 3.8 Load factors applied
- Excluded [ ] Included [ ]
- Values:

#### 3.9 Required life

#### 3.10 GEAR
- External [ ] Internal [ ] Without gear [ ]

#### 4. External S.R.
- Geometry
- Pinion
- Loads

#### 4.2 Required Module
- Tangential load (kN)

#### 4.3 Number of teeth required
- Torque (kNm)

#### 4.4 Addendum modif. factor

### 5. CRITICAL ITEMS
- Dimensions
- Others:

#### 5.1 Classification
- Commission [ ] FEM [ ] LLOYDS [ ] API [ ] BV [ ] DNV [ ] Other

#### 5.2 Specification relative to the application:

### SPECIFIC REQUIREMENTS OF THE APPLICATION

#### 6. Environment
- Operating temperature:

#### 6.2 Vibrations, shocks loads
- Storing temperature:

#### 6.3 Acceleration
- Deceleration
- Inert. mom./Rotation axis
- Varied:

#### 7. QUANTITY
- Yearly requirements:
- Qty per delivery:

#### 8. REQUIRED DELIVERY TIME:

---

Information available at the Commercial Department.
SKETCH OF YOUR APPLICATION SHOWING THE LOAD POSITIONS
SLEWING RING FUNCTIONS

BEARING FUNCTION

The knowledge of loads and working conditions is necessary to allow us to design and dimension the "BEARING" function of the slewing ring i.e.: movement type, speed, accelerations, temperatures, environment, etc.

The transmission of loads from one raceway to another varies according to the nature of applied loads.

In order to calculate the ideal dimensions of the raceway, we define the load equivalent to all external efforts in the most stressed areas. These loads are corrected by factors according to the application, the usage, etc.

We distinguish between:
- the application factor \( K_A \)
- the usage factor \( K_U \)
- the safety factor \( K_S \)

THE APPLICATION FACTOR \( K_A \)

is a coefficient taking into account the application specificity in relation to the slewing ring element.

This factor is established from ROLLIX experience. It is defined in the table "APPLICATION FACTORS".

THE USAGE FACTOR \( K_U \)

is defined according to the particular operating conditions: vibrations, shocks, occasional or accidental overloading, etc. If no other value is specified, then the nominal value is taken as 1.

THE SAFETY FACTOR \( K_S \)

is defined from standardized criteria for applications which must meet specific regulations such as: FEM, LLOYDS, API... This generally has the value 1, as the designer of the machine must include the regulation factors in the calculation of the loads applied on the bearing.
# SLEWING RING FUNCTIONS

## BEARING FUNCTION

### APPLICATION FACTORS

<table>
<thead>
<tr>
<th>MACHINES</th>
<th>Average speed Rpm</th>
<th>Applications factors $K_A$</th>
</tr>
</thead>
<tbody>
<tr>
<td>Armament turret</td>
<td>1,5</td>
<td>1,5</td>
</tr>
<tr>
<td>Bucket</td>
<td>1,5</td>
<td>1,65</td>
</tr>
<tr>
<td>Cable shovel</td>
<td>2</td>
<td>1,65</td>
</tr>
<tr>
<td>Compacter</td>
<td>2,5</td>
<td>1,80</td>
</tr>
<tr>
<td>Concrete mixer</td>
<td>5</td>
<td>2,40</td>
</tr>
<tr>
<td>Concrete pump</td>
<td>1,5</td>
<td>1,65</td>
</tr>
<tr>
<td>Dragline</td>
<td>1,5</td>
<td>1,65</td>
</tr>
<tr>
<td>Dock crane</td>
<td>1</td>
<td>1,65</td>
</tr>
<tr>
<td>Fairlead</td>
<td>0,8</td>
<td>1,35</td>
</tr>
<tr>
<td>Fork-lift truck</td>
<td>1</td>
<td>1,35</td>
</tr>
<tr>
<td>Fork-lift wheel</td>
<td>1,5</td>
<td>1,50</td>
</tr>
<tr>
<td>Grabbing crane</td>
<td>1,5</td>
<td>1,80</td>
</tr>
<tr>
<td>Heavy winch</td>
<td>2</td>
<td>1,65</td>
</tr>
<tr>
<td>Hydraulic lift platform</td>
<td>1</td>
<td>1,50</td>
</tr>
<tr>
<td>Hydraulic shovel</td>
<td>2,5</td>
<td>2</td>
</tr>
<tr>
<td>Loading dock crane</td>
<td>1</td>
<td>1,65</td>
</tr>
<tr>
<td>Merry-go-round</td>
<td>5</td>
<td>2,40</td>
</tr>
<tr>
<td>Mine digging machinery</td>
<td>1,5</td>
<td>2</td>
</tr>
<tr>
<td>Mobile fixed boom crane</td>
<td>1</td>
<td>1,5</td>
</tr>
<tr>
<td>Mobile grapple crane</td>
<td>1</td>
<td>1,80</td>
</tr>
</tbody>
</table>

<table>
<thead>
<tr>
<th>MACHINES</th>
<th>Average speed Rpm</th>
<th>Applications factors $K_A$</th>
</tr>
</thead>
<tbody>
<tr>
<td>Mobile telescopic crane</td>
<td>1</td>
<td>1,65</td>
</tr>
<tr>
<td>Offshore crane</td>
<td>1</td>
<td>1,8</td>
</tr>
<tr>
<td>Post jib crane</td>
<td>1</td>
<td>1,35</td>
</tr>
<tr>
<td>Railway crane</td>
<td>1</td>
<td>1,50</td>
</tr>
<tr>
<td>Rapid rotation radar</td>
<td>5</td>
<td>2</td>
</tr>
<tr>
<td>Rapid rotation scanner</td>
<td>3,5</td>
<td>1,65</td>
</tr>
<tr>
<td>Robotics</td>
<td>3,5</td>
<td>1,65</td>
</tr>
<tr>
<td>Service deck crane</td>
<td>0,8</td>
<td>1,35</td>
</tr>
<tr>
<td>Settler (water and sewage treatment)</td>
<td>0,6</td>
<td>1,35</td>
</tr>
<tr>
<td>Slow rotation radar</td>
<td>1</td>
<td>1,35</td>
</tr>
<tr>
<td>Slow rotation radiology</td>
<td>1</td>
<td>1,35</td>
</tr>
<tr>
<td>Tower crane, slewing jib type</td>
<td>1</td>
<td>1,65</td>
</tr>
<tr>
<td>Tower crane, slewing tower type</td>
<td>1</td>
<td>1,80</td>
</tr>
<tr>
<td>Track hook crane</td>
<td>1,5</td>
<td>1,80</td>
</tr>
<tr>
<td>Truck crane</td>
<td>1</td>
<td>1,50</td>
</tr>
<tr>
<td>Turntable</td>
<td>1</td>
<td>1,35</td>
</tr>
<tr>
<td>Vibrating compacter</td>
<td>2,5</td>
<td>2</td>
</tr>
<tr>
<td>Welding positionner</td>
<td>0,8</td>
<td>1,35</td>
</tr>
<tr>
<td>Windturbine</td>
<td>0,8</td>
<td>1,65</td>
</tr>
</tbody>
</table>

These factors are determined statistically and are based on a large number of observations for each type of application.

The standard parameters retained are as follows:
- Theoretical service life: 6000 hours.
- Work under normal weather conditions.
- Conventional application (and not specific).
SLEWING RING FUNCTIONS

BEARING FUNCTION

Selection of the ring according to capacity

The load capacity of the slewing ring is calculated according to its performance in function of:
- its geometric envelope,
- the nature of the ring materials,
- the heat treatment carried out,
- the nature, the number and the dimension of the rolling elements,
- the contact parameters of the rolling elements.

The curve of the maximum permissible capacity is drawn on a graph whose Ox axis bears the equivalent axial load and the Oy axis bears the tilting moment. To simplify it, it is represented by a straight line called the "LIMIT CURVE".

The ring size is determined by plotting the representative point of loads onto this curve. This point, called "application point" has the following coordinates:
- on the horizontal axis: \( P_x = F_{eq} \cdot K_A \cdot K_U \cdot K_S \)
- on the vertical axis: \( P_y = M_T \cdot K_A \cdot K_U \cdot K_S \)

In any case, the application point \( P \) must be under the limit curve.

Service life

Many external factors have a very important influence on the service life of the bearing. Among others, we can cite:
- geometric quality of supports,
- structure deformation under load,
- climatic conditions and environment,
- quality of operating maintenance
- conditions of use: repeated exposures to shocks, vibrations or sudden or intermittent movements can considerably reduce the theoretical service life.

\( K_T = \frac{O_L}{O_P} \)

An estimate of the theoretical service life can be obtained by comparing the application point to the limit curve: the ratio \( O_L/O_P \) is called \( K_T \).
**SLEWING RING FUNCTIONS**

**BEARING FUNCTION**

**Service Life**
An estimate of the service life $D$ can be obtained using the opposite graph:
- The curve indicates the estimated service life (hours) directly from the $K_T$ value on the horizontal axis.

**Rotational speed influence**
The service life $D$, estimated on the graph is only valid for the applications having a low rotational speed: 1 RPM. The value obtained must be multiplied by the speed factor $K_V$ indicated on the opposite graph when speeds differ from this.

$$D(n) = K_V \times D$$

For applications having oscillating movements, the following formula applies:

$$n_{\text{average}} = 0.60 \times n_{\text{real}}$$
SLEWING RING FUNCTIONS

FASTENING FUNCTION

In order to transmit the loads previously defined, it is necessary to realize an adequate mechanical fastening of the bearing on the associated frames, thus forming rigid connection of the ring and its supports.

While several fastening methods are feasible, the most efficient one remains the use of screws and nuts. Welding operations are absolutely prohibited.

The slewing ring proper functioning and the application safety are dependent on the correct bolting definition and fastening method during installation, complying with our workmanship.

External hexagon head screws must be preferred to cap screws (internal hexagon) whenever possible.

ROLLIX recommendation: screws and nuts, with guaranteed mechanical properties, matched, prelubricated so as to obtain a known and permanent screw/nut friction factor. The surface coating on the bolts must not generate any embrittlement.

**BOLTS QUALITY**

The ISO 898-1 standards define the bolting grade adapted to structure assemblies such as slewing rings. ROLLIX recommends the use of HIGH TENSILE bolts grade 10.9 and exceptionally grade 8.8 or 12.9 with rolled threads after heat treat.

The nuts must be of a same or higher grade as the associated screw. For a screw diameter d, a nut height of 1.25 d is recommended.

For rings in normalized steel Z or N, the use of hardened flat washers is required.

Minimal properties should be:

- a yield strength greater than or equal to 600 Mpa,
- a diameter: \( D_R = 2 \, d \),
- a thickness: \( h \geq 0.3 \, d \)

**Minimal mechanical characteristics (according to ISO)**

<table>
<thead>
<tr>
<th>GRADE</th>
<th>TENSILE (MPa)</th>
<th>YIELD (MPa)</th>
<th>FATIGUE (MPa)</th>
</tr>
</thead>
<tbody>
<tr>
<td>8.8</td>
<td>800</td>
<td>640</td>
<td>40</td>
</tr>
<tr>
<td>10.9</td>
<td>1040</td>
<td>940</td>
<td>40</td>
</tr>
<tr>
<td>12.9</td>
<td>1220</td>
<td>1100</td>
<td>40</td>
</tr>
</tbody>
</table>

*Exceptional*  
*Recommended*
SLEWING RING FUNCTIONS

FASTENING FUNCTION

BOLTING CALCULATION

The ROLLIX calculation formulae take into account the current standards and regulations as well as the many research and experimental findings. These calculations are mainly inspired by the AFNOR FD E 25.030, the recommendation VDI 2230 (1988) and the standard API 2C (1995).

Supported loads must be distinguished from hanging (suspended) loads acting in tension.
Consult ROLLIX, in the case of hanging (suspended) loads.

Standard calculation hypotheses
• Supported loads acting in compression.
• Equispaced bolts; i.e. equally positioned on the pitch circles.
• Steel rings and supports.
• Supports complying with our instructions: thickness, stiffness, surface evenness (see chapter STRUCTURES page 42).
• Rings bolted directly onto its supports.
• In cases of heavy radial loads, we recommend to use pilots or to glue so that bolts will not be subjected to shear stresses.
• The clamping length must be at least equal to five times the diameter: \( L_K \geq 5 \cdot d \).
SLEWING RING FUNCTIONS

FA STENING FUNCTIONS

Calculation of the number of bolts
When the ring has been previously selected according to its utilization and its load capacity, the bolting is then determined to correspond to the bearing capacity. The calculation of the minimum number of fasteners is carried out according to the following formula for the most unfavourable load case. In any case, a sufficient number of bolts ensuring an effective connection between ring and support frames must be kept, in order to avoid any ring deformation.

\[ N = \frac{1.6 \cdot F_k \cdot (4 \cdot M_T - F_A \cdot D_f)}{D_f \cdot (T_S - F_{pc})} \]

where:
\( N \) = Number of bolts theoretically necessary.
1.6 = Tightening factor (assembly error factor) for torque wrench Grade B according to FD E 25-030.
\( F_k \) = Bolt stretch factor, see sketch.
\( M_T \) = Total tilting moment applied to the ring in kNm.
\( F_A \) = Axial load in kN.
\( D_f \) = Fastener pitch circle diameter in m.
\( T_S \) = Tightening tension.
\( d \) = Bolt diameter in mm.
\( \Omega \) = Raceway mean diameter in m.
\( F_{pc} \) = Loss of tension due to embedding in kN, see graph.
\( L_k \) = Clamping length in mm.

Bolt stretch factor \( F_k \)
This factor takes the assembly geometry into account. It is based on the bolt diameter and the ratio of clamping length to diameter. The best fastening is obtained with through-holes in the ring and the supports: by using screws and nuts, the clamping length is long, bolt stiffness is satisfactory and tension losses are minimised. In case of screw fastening into tapped holes, the setting depth must not be less than 1.25\( \cdot d \).
SLEWING RING FUNCTIONS

FASTENING FUNCTIONS

Tightening tension: $T_s$

Tightening tension of fastening bolts must be sufficient to warrant the absence of looseness which is essential to ensure the resistance of the assembly fatigue.

Calculation of the minimum fastener preload:

It is useful to check that the standardized preload of the chosen bolt diameter is sufficient compared to the dynamic stresses imposed when operating.

$$T_s > \left( \frac{2.25}{N} \right) \left( \frac{4.\Delta T}{\phi} \right) - FA + 80 \text{ N.d.10}^{-3}$$

The standardized tightening tension at 80% of Re must be selected from the following table according to the chosen bolt diameter: bolting grade 10.9.

<table>
<thead>
<tr>
<th>Diameter (mm)</th>
<th>12</th>
<th>14</th>
<th>16</th>
<th>20</th>
<th>22</th>
<th>24</th>
<th>27</th>
<th>30</th>
<th>33</th>
</tr>
</thead>
<tbody>
<tr>
<td>Tension (kN)</td>
<td>56</td>
<td>77</td>
<td>106</td>
<td>166</td>
<td>208</td>
<td>239</td>
<td>315</td>
<td>385</td>
<td>480</td>
</tr>
</tbody>
</table>

Loss of tension:

During tightening and under external loads, peening of the surface roughness of the contact parts occurs, reducing the initial bolts elongation and thus producing a loss of tension, which decreases the preload in the assembly. This loss of tension has been tabulated on the following graph which shows values in function of diameter $d$ and the ratio $L_k/d$.

Calculation of under head contact pressure:

Usually, this calculation is not required when treated flat washers are used. It is however recommended when cap screws are used. (Chc).

We must have:

$$\frac{F_{B \max}}{A_c} < P_{adm}$$

with

$$F_{B \max} = T_s + 0.13 FE$$

and

$$A_c = \left( \frac{\pi}{4} \right) \left( d_w^2 - D_i^2 \right)$$

Allowable pressure:

- for steels N and Z: 400 MPa
- for steels D and X: 620 MPa
- for steels type E36: 270 MPa

IMPORTANT REMARK

The use of elastic washers whatever the type or model is absolutely prohibited and will void all warranty.
ROLLIX slewing rings generally incorporate a SLEWING mechanism to control rotation of the mobile part. This function can be achieved by various means:

1. gear drive (the most frequent case)
2. belt drive
3. chain drive
4. direct drive

GEOMETRY
Most of the ROLLIX bearings have a gear improved by **positive addendum modification** which notably decreases pressures as well as by a **truncation** avoiding teeth root interference at the pinion. It is also essential to make a positive addendum modification on the pinion teeth, in order to avoid the geometric interference which appears under 18 teeth.

Furthermore, the drive stresses induce shaft bending and gear deformation which are harmful to good meshing. To prevent these faults, we recommend profile corrections be carried out on the pinions: i.e. crowning and tip relief. Our Engineering Department will assist upon request.

RESISTANCE
Our rating graphs indicate the values of allowable maximum tangential force in fatigue \( T \).

The continuous operating capacity is obtained by the application of a suitable load moderating factor:

\[
T = 2 \frac{C_d}{D_{ref}}
\]

\( C_d \) = Torque on gear
\( D_{ref} \) = Reference diameter

Unless otherwise stated, the indicated values are valid for geared rings made of normalized steel XC45: code Z.

When these values are not sufficient, ROLLIX can proceed with contour hardening which considerably improves the resistance to tooth root bending and the resistance to contact pressure. For very heavy loading conditions, ROLLIX carries out complete hardening of the tooth and of its root in the wheel rim. When only a better wear resistance is required, surface hardening of tooth flanks only is possible.
SLEWING RING FUNCTIONS

SLEWING FUNCTION

GEAR QUALITY
Unless stated otherwise, ROLLIX manufactures slewing ring gears according to AFNOR or DIN standards which meet the following criteria:

WITHOUT SUPERFICIAL HEAT TREATMENT

<table>
<thead>
<tr>
<th>DIN class</th>
<th>AFNOR class</th>
<th>Maximum diameter</th>
<th>Maximum module</th>
<th>Options</th>
</tr>
</thead>
<tbody>
<tr>
<td>12</td>
<td>12</td>
<td>all sizes</td>
<td>25</td>
<td>Module 45 with special tooling</td>
</tr>
<tr>
<td>10-11-12</td>
<td>11-10-9</td>
<td>all sizes</td>
<td>20</td>
<td>Specific equipment necessary</td>
</tr>
<tr>
<td>9-10</td>
<td>8-9</td>
<td>3100</td>
<td>22</td>
<td>• made upon request</td>
</tr>
<tr>
<td>7-8</td>
<td>7</td>
<td>2500</td>
<td>20</td>
<td>When a higher gear quality such as grade 5 or 6 is needed, gear grinding becomes necessary (ask our Engineering department).</td>
</tr>
</tbody>
</table>

SUPERFICIALLY HARDENED GEAR

• Generally, by contour hardening to 55 HRc (± 5).

• The gear classes stated above are offset and ROLLIX can meet AFNOR or DIN standards for grades 11-12.

IMPORTANT
ROLLIX considers that the relevant gear characterizing parameters for each quality class defined by AFNOR, DIN or ISO must be met. In case a customer does not require all of the parameters and needs to meet only one or two of them, ROLLIX can achieve higher qualities.
OTHER FEATURES

PRECISION - TOLERANCES

Standard ring tolerances are generally defined according to ISO 286-1 and 2 standards.

For applications requiring a higher precision level: robotics, radars, etc., a better grade is considered. The tolerance values are then indicated on the bearing drawing. For bearings of large diameter with thin cross sections where radial stiffness is low, the tolerance values must be considered when bearing is assembled to its supports, these ensuring proper circularity.

GEOMETRY
The selected criteria are:
• For the diameters: Js 13
• For the centerings
  Bores: H9
  Shafts: f9
• For the overall height: ± 1 mm

FASTENING
The bolt circle diameters are machined to tolerance Js10 with a minimum of ± 0.2 mm.

GENERAL TOLERANCES TABLE (ACCORDING TO ISO 286-2)

<table>
<thead>
<tr>
<th>Diameter Js10</th>
<th>Pilot H9</th>
<th>Spigot f9</th>
</tr>
</thead>
<tbody>
<tr>
<td>from 180 to 250 (in mm)</td>
<td>+115</td>
<td>+60</td>
</tr>
<tr>
<td>250 to 315</td>
<td>+130</td>
<td>+62</td>
</tr>
<tr>
<td>315 to 400</td>
<td>+140</td>
<td>+68</td>
</tr>
<tr>
<td>400 to 500</td>
<td>+155</td>
<td>+76</td>
</tr>
<tr>
<td>500 to 630</td>
<td>+175</td>
<td>+80</td>
</tr>
<tr>
<td>630 to 800</td>
<td>+200</td>
<td>+86</td>
</tr>
<tr>
<td>800 to 1000</td>
<td>+230</td>
<td>+98</td>
</tr>
<tr>
<td>1000 to 1250</td>
<td>+260</td>
<td>+110</td>
</tr>
<tr>
<td>1250 to 1600</td>
<td>+310</td>
<td>+120</td>
</tr>
<tr>
<td>1600 to 2000</td>
<td>+370</td>
<td>+130</td>
</tr>
<tr>
<td>2000 to 2500</td>
<td>+440</td>
<td>+145</td>
</tr>
<tr>
<td>2500 to 3150</td>
<td>+540</td>
<td>+165</td>
</tr>
</tbody>
</table>

<table>
<thead>
<tr>
<th>Diameter Js13 (in mm)</th>
<th>± 0.36</th>
<th>± 0.405</th>
<th>± 0.55</th>
</tr>
</thead>
<tbody>
<tr>
<td>from 180 to 250 (in mm)</td>
<td>± 0.36</td>
<td>± 0.405</td>
<td>± 0.55</td>
</tr>
<tr>
<td>250 to 315</td>
<td>± 0.445</td>
<td>± 0.55</td>
<td>± 0.625</td>
</tr>
<tr>
<td>315 to 400</td>
<td>± 0.485</td>
<td>± 0.625</td>
<td>± 0.70</td>
</tr>
<tr>
<td>400 to 500</td>
<td>± 0.825</td>
<td>± 0.70</td>
<td>± 0.975</td>
</tr>
<tr>
<td>500 to 630</td>
<td>± 1.15</td>
<td>± 0.975</td>
<td>± 1.4</td>
</tr>
<tr>
<td>630 to 800</td>
<td>± 1.65</td>
<td>± 0.975</td>
<td>± 1.65</td>
</tr>
</tbody>
</table>

GEAR
The value of the total run-out is indicated in the drawing title block. The measurement and tolerance over K teeth are also indicated on the drawing. This dimension includes the contribution of the ring to the meshing backlash.

BEARING
• The AXIAL RUN-OUT of the supporting faces is measured using a dial gauge over one full revolution (see opposite sketch).
OTHER FEATURES

PRECISION - TOLERANCES

- THE RADIAL RUN-OUT of the spigots is also measured by rotation (following sketch).

These measurements are carried out by placing the magnetic base of a dial gauge on the fixed ring, while the filler is taken in contact with the element to be measured. The reading will be obtained during one rotation of the turning ring (value T.I.R).

- THE DEFLECTION UNDER MASTER LOAD is measured as follows:

One of the bearing ring is fastened to a rigid support and a master load $F$ is applied axially on the other ring to a point closely located to the raceway. The deflection under load $F$ is measured using a dial gauge whose base is attached to the other ring. The deflection under standard load of every slewing ring is measured at the factory. This value is compared to the allowable limits, and is registered.

DEFLECTION UNDER MASTER LOAD
OTHER FEATURES

SLEWING TORQUE

Calculation of the necessary torque to enable a rotation of the whole assembly takes into account:
- loads on the machine,
- rotating masses,
- distance of these masses to the rotation axis,
- speeds and accelerations,
- resisting torques.

Two types of torques are distinguished:

Start up slewing torque: \( Cd = Crv + Crc \)

Acceleration slewing torque: \( Cg = Crv + Crc + Ca \)

\( Crv \) = Friction torque of unloaded bearing

\( Crc \) = Rotating torque due to loads

\( Ca \) = Acceleration torque

\( Cd \) = Starting torque

All these torques are expressed in kNm.

\( Crc \): ROTATION TORQUE DUE TO LOADS

The starting torque required takes into consideration loads on the bearing and friction of the components.

Balls type slewing ring

\[
Crc = \left[ \frac{13,11 \, MT}{\Omega \, m} + 3 \, F_A + 11,34 \, F_R \right] \, \Omega \, m \cdot 10^{-3}
\]

Crossed rollers type slewing ring:

\[
Crc = \left[ \frac{15,3 \, MT}{\Omega \, m} + 3,75 \, F_A + 8,19 \, F_R \right] \, \Omega \, m \cdot 10^{-3}
\]

\( MT \) = Resulting moment in kNm

\( \Omega \, m \) = Raceway mean \( \Omega \) in meters

\( FA \) = Axial load in kN

\( FR \) = Radial load in kN

\( Ca \): ACCELERATION TORQUE

The torque needed to accelerate the loads from the initial speed up to the final speed, during time \( t \), is defined by:

\[
Ca = \frac{\pi \cdot n \cdot I}{30 \cdot t} \cdot 10^{-3}
\]

\( t \) = Acceleration time in sec.

\( n \) = Speed variation in RPM

\( I \) = Moment of inertia of the machine in Kg \( \cdot m^2 \)

\( I = I_1 + I_2 + I_3 + \ldots + I_n \)

where \( I_1 \ldots I_n \) = moments of inertia of the moving loads with regard to the rotation axis expressed in Kg \( \cdot m^2 \).

Generally we have:

\( I_1 = G_1 \times r_1^2 \)

\( I_n = G_n \times r_n^2 \)

\( G_1 \ldots G_n \) = Mass of various rotating components expressed in Kg.

\( r_1 \ldots r_n \) = Distances between the loads centre of gravity and the ring rotation axis expressed in meters.

Note: The resisting torque depends on the support surface flatness and lubrication.
OTHER FEATURES

SLEWING TORQUE

The friction torque of standard slewing rings is defined in the following graph. ROLLIX, upon request, can supply slewing rings with lower or higher torque values.

LOADS APPLIED ON THE RING

Axial $F_A : 68 \text{kN} + 5 \text{kN} = 73 \text{kN}$
Radial $F_R : 0.29 \text{kN}$, negligible
Moment $M_T : 5 \text{kN} \times 1.5 \text{m} = 7.5 \text{kNm}$

SLEWING TORQUE : Raceway mean $\varnothing = 2$ meters

Crv : according to the graph : $1 \text{kNm}$

\[
C_{rc} = \left[ \frac{13.11 \times 7.5}{2} + (73 \times 3) + (11.34 \times 0) \right] \times 10^{-3}
\]

$C_{rc} = 0.536 \text{kNm}$

Slewing torque at start up

\[
C_d = 1 + 0.536 = 1.536 \text{kNm}
\]

Platform moment of inertia :

\[
\frac{M}{2} = \frac{6800 \times 2^2}{2} = 13600 \text{Kg m}^2
\]

Cubic moment of inertia :

\[
M_{r^2} = 500 \times 1.5^2 = 1125 \text{Kg m}^2
\]

Total moment of inertia :

\[
13600 + 1125 = 14725 \text{Kg m}^2
\]

Acceleration torque :

\[
n = 6 - 2 = 4 \text{ RPM}
\]

\[
C_a = \frac{14725 \times \pi \times 4}{30 \times 20} \times 10^{-3} = 0.3084 \text{kNm}
\]

Slewing torque during acceleration

\[
C_g = 1 + 0.536 + 0.3084 = 1.845 \text{kNm}
\]

APPLICATION EXAMPLE

Platform diameter : 4 m.
Platform mass : 6800 kg
Cube mass : 500 kg
Ball type slewing ring
raceway mean $\varnothing : 2$ m.
Distance from the cube to the rotation axis : 1.5 m.
Initial speed : 2 RPM
Final speed : 6 RPM
Acceleration time : 20 sec.
OTHER FEATURES

PROTECTION

SEALS
ROLLIX slewing rings are generally equipped with protecting seals on both sides of the raceway. These seals have the function of:
• protecting raceways against small-sized contaminating agents,
• retaining the lubricant in the raceway.
For specific requirements, ROLLIX designs suitable protective devices, such as:
• standard and specific seals,
• lip seals
• combination of two or more seals.

SHROUDS
For severe application conditions and in order to limit the effects of aggressive agents such as:
• swarf and metal chips
• welding grains
• abrasives
• mud
• sand
• water and heavy sea splashes
• cutting fluids...

ROLLIX strongly recommends to install effective protective shields. Furthermore, during cleaning with solvents or pressurised water, it is advisable to avoid working in the protective device area.

SURVEY
If, during regreasing, large grease discharges are noticed, it is advisable to check:
• that seal is still correctly positioned,
• that seal is not damaged (cut, torn, worn),
• that seals are still capable for proper bearing operation.
Therefore, this seal can either be reinstalled or replaced.

SEAL RANGE TABLE

<table>
<thead>
<tr>
<th>Operating conditions</th>
<th>Ranges</th>
</tr>
</thead>
<tbody>
<tr>
<td>&quot;Normal&quot; : - 30° C à + 70° C</td>
<td>NITRILE-BASED elastomer</td>
</tr>
<tr>
<td>&quot;Extreme&quot; : θ &lt; -30°C ; 70°C &lt; θ &lt; 200°C</td>
<td>FLUOR-BASED elastomer</td>
</tr>
<tr>
<td>&quot;Special&quot; : Various physical or chemical aggressive agents.</td>
<td>NITRILE-BASED elastomer modified or others</td>
</tr>
</tbody>
</table>

PROTECTIVE SURFACE COATING

For particular applications, ROLLIX suggests a protective surface treatment, such as:

1. Zinc plating (+ chromate coating)
2. Phosphate coating
3. Chemical nickel plating
4. Paint
5. Miscellaneous treatments, for example, chrome plating, Schoop’s metal spraying process, anodic oxidization, etc.

Consult us.
INSTALLATION AND MAINTENANCE

- MOUNTING STRUCTURES
- INSTALLATION
- MARKING
- MAINTENANCE
- UTILIZATION LIMITS
STRUCTURES

SUPPORT STRUCTURES

CHASSIS DESIGN
The slewing ring has a moderate axial stiffness: the diameter is large compared to the cross section. It must be mounted on a machined supporting base, ensuring sufficient stiffness with regard to the loads to be transmitted. This makes it possible to ensure an even distribution of stresses and to avoid any deformation during operation, which would be harmful to the good working of the ring.

Therefore, it is necessary to use supporting bases with a minimum thickness not less than the values indicated in the table above. The width of the supporting surfaces is to be at least equal to that of the ring. We recommend structural reinforcements in a circular frame placed directly beneath the raceway. For better loading uniformity, thick circular frames are preferred to thin reinforcements with ribs.

<table>
<thead>
<tr>
<th>Raceway mean diameter (mm)</th>
<th>500</th>
<th>750</th>
<th>1000</th>
<th>1250</th>
<th>1500</th>
<th>2000</th>
<th>2500</th>
<th>3000</th>
</tr>
</thead>
<tbody>
<tr>
<td>Minimum thickness (mm)</td>
<td></td>
<td></td>
<td>25</td>
<td>30</td>
<td>35</td>
<td>40</td>
<td>50</td>
<td>60</td>
</tr>
</tbody>
</table>

SHAPE TOLERANCES
Shape defects of the supports lead to deformations of the raceway. This can cause tight spots or possible seizure and will reduce the bearing service life. The maximum flatness defects must not exceed the values of the opposite table.

<table>
<thead>
<tr>
<th>Raceway mean diameter (mm)</th>
<th>500</th>
<th>750</th>
<th>1000</th>
<th>1250</th>
<th>1500</th>
<th>2000</th>
<th>2500</th>
<th>3000</th>
</tr>
</thead>
<tbody>
<tr>
<td>Maximum tolerance Crossed Roller (mm)</td>
<td>0,10</td>
<td>0,12</td>
<td>0,15</td>
<td>0,18</td>
<td>0,20</td>
<td>0,25</td>
<td>0,29</td>
<td>0,32</td>
</tr>
<tr>
<td>Maximum tolerance Balls (mm)</td>
<td>0,12</td>
<td>0,18</td>
<td>0,21</td>
<td>0,25</td>
<td>0,28</td>
<td>0,33</td>
<td>0,38</td>
<td>0,42</td>
</tr>
</tbody>
</table>
STRUCTURES

SUPPORT STRUCTURES

These maximum values are allowable for "long waves" in the circumferential direction. "Short waves" for example between two fastening holes, must not exceed 1/4 of the values in the opposite table. Waves in the radial direction (conicity) must be less than 0,05 mm / m of raceway mean Ø.

STIFFNESS TOLERANCES

The stiffness of the supporting frames must be such that deflections do not exceed the values of the opposite table under maximum load.

PLASTIC CEMENTS

If the above tolerances could not be met, then Epoxy-type resins can be used instead of finish-machining of the surfaces.

Several products are possible depending on the dimensions and the amount of out of flatness to be compensated. Detailed instructions are available from our Sales Department.
INSTALLATION

TRANSPORT - HANDLING

Our slewing rings are carefully packed in order to avoid any damage during transport.
Transport and storage are to be carried out in horizontal position only; transport in other positions requires special methods.
As with any mechanical precision part, the rings must be handled with care avoiding any shocks, particularly along the radial axis.
Handling should be carried out with suitable equipment for the weight of the part, which is indicated on the identification tag.

DELIVERY - STORAGE

The packed rings have an anticorrosive surface protection allowing a 6-month-storage in a covered and temperate room.
A suitable protection must be applied for longer storage.
It is necessary to regrease after each 18-month-period (see paragraph MAINTENANCE - LUBRICATION).

UNPACKING - PREPARATION

When unwrapping the bearing:
• Take care not to cut the protective seals when removing the packing paper.
• Cut this paper, preferably on the external diameter, and not on the upper or lower faces.
When degreasing the bearing:
• Use a standard commercially available solvent. Chlorine containing solvents are prohibited.
• Take care not to introduce any solvent under the seals or in the raceways.
• Before fitting the grease nipples or junction pipes, remove the plastic caps or the Hc screws from the greasing holes.

INSTALLATION

WHEN INSTALLING THE BEARING:

Structure
• Make sure that supporting structures comply with specifications (see chapter STRUCTURES).
• Check for chips, weld seam particles, corrosion signs, etc.
• Check the good mating of the rings on the supports.

SPIGOTS

When loads along the radial axis are important, especially when the bearing is placed vertically, it is then mandatory to use the centerings provided for that purpose.
The intercalative structural adhesive type LOCTITE 586 provides a good means to limit the relative displacements between rings and supports.
See our technical instructions IT ETR 521, available upon request.

POSITIONING

The hardening junction which is marked by a red line on the geared ring, and located at the filler plug on the other ring, must be placed at 90° to the main load axis or to the arm supporting the load (see chapter MARKING).
INSTALLATION

INSTALLATION

Fastening
- Check that fasteners are really of the recommended grade, e.g. marked 10.9 on the head and that threads are properly lubricated.
- For bearings in normalized steel Z or N, the use of treated hardened flat washers is required as follows:
  - the yield strength is greater than or equal to 600 Mpa,
  - the diameter DR = 2 d,
  - the thickness h > 0.3 d.
The elastic washers type Belleville, Grower or others of whatever type or pattern are absolutely prohibited and will void all warranty.
- Install all fasteners and tighten lightly.
- Then tighten to the specified value using a properly calibrated torque wrench; hydraulic devices are advisable.
- Use a "star shape" pattern when torquing which ensures a uniform tightening over the whole periphery. The tightening torque to be applied is defined in the following table for bolt grade 10.9 and a screw/nut friction factor of 0.12 according to VDI 2230.

<table>
<thead>
<tr>
<th>Diameter (mm)</th>
<th>10</th>
<th>12</th>
<th>14</th>
<th>16</th>
<th>18</th>
<th>20</th>
<th>22</th>
<th>24</th>
<th>27</th>
<th>30</th>
<th>33</th>
</tr>
</thead>
<tbody>
<tr>
<td>Torque (N.m)</td>
<td>69</td>
<td>117</td>
<td>185</td>
<td>280</td>
<td>390</td>
<td>560</td>
<td>750</td>
<td>960</td>
<td>1400</td>
<td>1900</td>
<td>2600</td>
</tr>
</tbody>
</table>

Checking the gear backlash

WHEN INSTALLING THE PINION:

Gear
- The pinion should be located approximately at 90° of the major loading axis.
- Adjust the driving pinion to the maximum eccentric point of the ring gear, marked by a blue line.
- At this stage, the backlash must be within the limits of the calculated values or minimum at 0.05 x module.
- When several pinions are used, each one must be adjusted to the same conditions.
- During tests, make sure that good alignment of the pinion and of the slewing ring axes permits a satisfactory contact across all the gear width.
- Before running, lubricate the teeth of the slewing ring gear and of the pinion (see chapter MAINTENANCE).
INSTALLATION

Tests - inspections
After final tightening of all fasteners :
• Rotate the ring for at least 3 turns.
• Recheck the backlash value on the gear mesh over one full revolution.
• Measure the total deflection under a known load. The checked points should be marked.
It is advisable to register these values in a maintenance logbook specific to the machine, see our "COMMISSIONNING DOCUMENTS" ref. IT-DTR.19.005.

Operating process :
• Place a measuring device between the two rings as close as possible to the raceway on the main load axis : precision grade 0,1 mm minimum.
• Calibrate to zero, under a known load.
• Apply a measured load.
• Read the deflection value at the point considered.
• Repeat this measurement in the previously marked points.
• Take into account the deflection of the support chassis and the fasteners elongation corresponding to the instrument position.
MARKING

IDENTIFICATION TAG

Each ring is individually identified by a metallic tag riveted close to the filler plug on the ungeared ring.

This tag states:
- manufacturing date,
- part number of the ring,
- its serial number,
- its weight in kilograms.

INSTALLATION MARKS

In order to permit the correct installation of the bearing, ROLLIX slew rings are marked as follows:

Hardening junction:
- Located on the filler plug for the ungeared ring.
- Marked by a red line on the gear face for the geared ring.
These areas must be placed as close as possible to the neutral axis of the loads: ideally at zero moment axis.

Gear run-out:
The maximum point of the gear run-out is marked by:
- Two blue lines on the corresponding tips of the teeth,
- a O stamped on the face opposite the mounting surface.
The backlash of the driving pinion must be adjusted at this point.
MAINTENANCE

MAINTENANCE - LUBRICATION

A suitable lubrication is essential for the longevity of the raceways and gears. The operating conditions such as loads, temperatures, speeds, vibrations, etc... determine the choice of lubricant.

RACEWAY

Unless otherwise specified, the slewing rings are delivered greased.
Standard grease : ESSO BEACON EP 2 or equivalent.

Main properties required for multi-purpose grease :
As being a significant component of the bearing, grease will improve the bearing capabilities and longevity.

Recommendations for bearing lubricant :
- Lithium-base soap.
- Minimum viscosity of the base : 150 mm²/sec.
- Grade NLGI 2.
- Anti-wear and extreme pressure additives.
- Service temperature :
  - 30°C to +120°C
- 4 ball test : weld load : ASTM D 2596 (NT24) > 300
- Maximum NDM :
  for balls ≈ 60 000
  for rollers ≈ 30 000

GEAR

A protection against oxidation is applied.

GREASING HOLES

Radially or facially located, depending on design, these holes are generally tapped M10 x pitch . 1.00 and closed by plastic caps or Hc screws.
Remove these plugs before fitting the slewing ring with grease nipples or linked to a centralized lubrication system.

Caution : the filler plug for the rolling elements has a blind tapped hole which is not a greasing hole.

REGREASING METHODS

Whenever the application allows it, greasing must be carried out during rotation at slow speed, on two revolutions minimum, through all the greasing holes.

GREASING FREQUENCY

Raceway and gear.
The greasing frequency varies according to utilization and environment. We recommend regreasing every 150 hours in normal usage. This frequency is to be reduced to 50 hours when the conditions of application are severe or if the environment is dusty or wet.
Greasing is required, before and after a long idle period.
Regrease every 6 months, while rotating, during prolonged idle periods.

GREASE QUANTITY

Raceway :
The grease quantity is defined by the Engineering Department whenever a detailed bearing calculation is provided.
Approximate practical formula to determine the minimum necessary quantity "Q" in cm³ :

\[
Q = 0.005/3 \times D \times H
\]

D = raceway mean Ø of the bearing in mm.
H = overall height of the ring in mm.
In all cases, a light extrusion of new grease must appear at the protection seal lips.

Gear :
The grease must entirely cover the flanks of the pinion and of the ring gear whether applying by brush or spraying.

SPECIFIC APPLICATIONS

Upon request, the ROLLIX Engineering Department will provide solutions for extreme conditions : temperature, speed, etc.
See our data sheet IT ETR 551.
MAINTENANCE

MAINTENANCE - LUBRICATION

CORRESPONDENCE TABLE
According to our experience, the greases mentioned in the opposite table are compatible with each other and with the components of the bearings.

It is possible to use other lubricants provided that you are sure of their compatibility with the ROLLIX standard recommendation beforehand.

Greases containing molybdenum disulphide MoS\textsubscript{2} are strictly forbidden.

<table>
<thead>
<tr>
<th>BEARING</th>
<th>GREASE BRAND</th>
<th>GEAR</th>
</tr>
</thead>
<tbody>
<tr>
<td>Aralub HLP2</td>
<td>ARAL</td>
<td>Aralub LFZ1</td>
</tr>
<tr>
<td>Rhus L 474/2</td>
<td>MOTUL/BECHEM</td>
<td>Berulit GA 400</td>
</tr>
<tr>
<td>Energol E5 - EP2</td>
<td>BP</td>
<td>Energol WRL/GR 154 GS</td>
</tr>
<tr>
<td>Grease LMX</td>
<td>CASTROL</td>
<td></td>
</tr>
<tr>
<td>Epexa 2 / Epexelf 2</td>
<td>ELF</td>
<td>Cardrex DC1</td>
</tr>
<tr>
<td>Beacon EP2</td>
<td>ESSO</td>
<td>Surré Fluid NX</td>
</tr>
<tr>
<td>Mobilux EP2</td>
<td>MOBIL</td>
<td>Mobilgear OGL 007</td>
</tr>
<tr>
<td>Retina EP2 - Alvania EPLF2</td>
<td>SHELL</td>
<td>Malléus GL 205</td>
</tr>
<tr>
<td>Multis EP2 - Lical EP2</td>
<td>TOTAL FINA ELF</td>
<td>Ceran AD</td>
</tr>
</tbody>
</table>

This table is subject to change in accordance with the manufacturers’ research works.

PREVENTIVE MAINTENANCE

PROTECTION SURVEY
A visual examination makes it possible to ensure the integrity of the protective seals:
• absence of excessive stretch or rips,
• correct positioning,
• wear of the friction lip.

If necessary, replace the seal.

After regreasing, wipe clean residue of old grease and check for pollutants such as sand, coal, metallic particles, etc.

FASTENER SURVEY
It is particularly important to check that the required preload level of the bolts is still maintained as the fasteners of the slewing rings are essentially working in fatigue.

ROLLIX recommends to retighten the fasteners after the first two to four months of utilization and then proceed to a systematic yearly check.
If any bolt is found loose, a further in deep examination is essential. The necessary preservative measures must then be exercised.

Some regulations impose the replacement of fasteners every seven years or every 14 000 working hours.
In any case, refer to local rules and regulations enforced connected with the application.

ORIENTATION SURVEY
During cleaning prior to regreasing of the gear:
• Check carefully for any foreign body at the tooth root, ring and pinion.
• Check the even load distribution of the pinion on the entire width of the ring gear and correct the alignment of the axes if needed.
• Check the backlash value.
UTILIZATION LIMITS

CHECKING THE DEFLECTION UNDER LOAD

ROLLIX delivers its bearings with a preload ensuring proper functioning and optimum safety. During the product life, the preload decreases resulting in a noticeable increase of deflection under load. The bearing must be replaced when the deflection becomes incompatible with the proper functioning of the machine and with the required safety conditions for the type of material used.

ROTATION SURVEY

To quantify the wear factor, it is necessary to know the deflection under load.

- In new condition : \( J_0 \)
- At time of survey : \( J_1 \)

These measurements are made under the same initial conditions after having checked the tightening of fasteners (see chapter INSTALLATION, Tests - Inspection, page 46).

Wear is the difference : \( u = J_1 - J_0 \).

- The bearing must be placed under survey when : \( u \geq J_0 \).
- Its replacement must be considered when : \( u \geq 1.5 \ J_0 \).
- and it is required when : \( u \geq 2 \ J_0 \).

In any case, refer to laws and regulations in force pertaining to the application in the concerned country.

It is most advisable to register the measured values in the maintenance logbook specific to the machine.
RANGE OF PRODUCTS

- CODING SYSTEM
- BEARING FAMILIES
- DRAWINGS
- CAPACITY CURVES
- BEARING REFERENCES

- CROSSED ROLLERS External Gear 06
- CROSSED ROLLERS Internal Gear 07
- CROSSED ROLLERS Without Gear 08
- BALLS External Gear 01
- BALLS Internal Gear 02
- BALLS Without Gear 03
- LIGHT SERIES 21 to 29
- LIGHT SERIES SOLID SECTIONS 31 to 39
# CODING SYSTEM

ROLLIX bearings are designated by a part reference including numbers and letters, according to the opposite codes.

<table>
<thead>
<tr>
<th>Family</th>
<th>Mean raceway ball/roller pitch Ø</th>
<th>Variation ( N ) in the family</th>
<th>Material code letter ungeared ring</th>
<th>Material code letter geared ring</th>
</tr>
</thead>
<tbody>
<tr>
<td>06 1116 00 Z Z 1 2 A</td>
<td></td>
<td></td>
<td>Material code : Z Improved XC45 (or similar)</td>
<td></td>
</tr>
<tr>
<td></td>
<td></td>
<td></td>
<td>X XC45 core hardened</td>
<td></td>
</tr>
<tr>
<td></td>
<td></td>
<td></td>
<td>N 42CrMo4 normalized</td>
<td></td>
</tr>
<tr>
<td></td>
<td></td>
<td></td>
<td>D 42CrMo4 core hardened</td>
<td></td>
</tr>
<tr>
<td></td>
<td></td>
<td></td>
<td>K aluminium alloys</td>
<td></td>
</tr>
<tr>
<td></td>
<td></td>
<td></td>
<td>J other materials</td>
<td></td>
</tr>
</tbody>
</table>

<table>
<thead>
<tr>
<th>Gear treatment</th>
<th>Surface coating</th>
<th>Revision index: Project drawings : revisions according to numerical index from 1 to 9</th>
</tr>
</thead>
<tbody>
<tr>
<td>0 gear without heat treatment</td>
<td>0 oil storage protection</td>
<td>Project drawings : revisions according to numerical index from 1 to 9</td>
</tr>
<tr>
<td>1 gear with contour hardening on flanks and roots</td>
<td>1 zinc or cadmium plated + chromate coating</td>
<td>Project drawings : revisions according to numerical index from 1 to 9</td>
</tr>
<tr>
<td>2 gear flanks superficially hardened</td>
<td>2 phosphate coating</td>
<td>Project drawings : revisions according to numerical index from 1 to 9</td>
</tr>
<tr>
<td>3 other hardening treatments</td>
<td>3 chemical nickel plating</td>
<td>Project drawings : revisions according to numerical index from 1 to 9</td>
</tr>
<tr>
<td>4 paint : 40 ROLLIX standard primer</td>
<td>41 to 49 specific paint systems</td>
<td>Project drawings : revisions according to numerical index from 1 to 9</td>
</tr>
<tr>
<td>5 to 9 miscellaneous treatments (6 = anodic oxidization)</td>
<td></td>
<td>Project drawings : revisions according to numerical index from 1 to 9</td>
</tr>
</tbody>
</table>

Manufacturing drawings : first production = index A

subsequent revisions = index B to Z
# BEARING FAMILIES

## DEFINITION OF CLASSES

Classes I, III and V define the amount of internal preload of the bearing. Class V corresponds to the maximal preload.

This table shows the main families of ROLLIX slewing rings. Slewing rings of specific types are listed in the document IT ETR 002.

<table>
<thead>
<tr>
<th>Class</th>
<th>IV</th>
<th>Class</th>
<th>III &amp; V</th>
<th>Class</th>
<th>I</th>
<th>III</th>
<th>V</th>
<th>Class</th>
<th>I</th>
<th>III</th>
<th>V</th>
<th>Ext.</th>
<th>Int.</th>
<th>Without</th>
</tr>
</thead>
<tbody>
<tr>
<td>Single row</td>
<td>Double row</td>
<td>Single row</td>
<td>Double row</td>
<td>Normal clearance</td>
<td>Reduced clearance</td>
<td>Precision</td>
<td>Normal clearance</td>
<td>Reduced clearance</td>
<td>Precision</td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>06</td>
<td>16</td>
<td>01</td>
<td>11</td>
<td>21</td>
<td>24</td>
<td>27</td>
<td>31</td>
<td>34</td>
<td>37</td>
<td>●</td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>07</td>
<td>17</td>
<td>02</td>
<td>12</td>
<td>22</td>
<td>25</td>
<td>28</td>
<td>32</td>
<td>35</td>
<td>38</td>
<td>●</td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>08</td>
<td>18</td>
<td>03</td>
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<td>23</td>
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<td>36</td>
<td>39</td>
<td>●</td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
</tbody>
</table>

**FAMILY 06**

**FAMILY 07**

**FAMILY 08**

**FAMILY 16**

**FAMILY 17**

**FAMILY 18**

**FAMILY 01**

**FAMILY 02**

**FAMILY 03**

**FAMILY 11**

**FAMILY 12**

**FAMILY 13**

**FAMILY 21 - 24 - 27**

**FAMILY 22 - 25 - 28**

**FAMILY 23 - 26 - 29**

**FAMILY 31 - 34 - 37**

**FAMILY 32 - 35 - 38**

**FAMILY 33 - 36 - 39**
Each bearing is designated by a drawing reference, example: 06 1116 00.
The drawing defines both principal and functional dimensions: spigots, bolt patterns and dimensions, overall height, reference diameter of the gear.

**Fastening:**
- **40:** number of fastening holes,
- **“=”** symbolizes equispaced holes,
- **“#”** indicates non equispaced holes,
- **1035:** bolt circle diameter.

The symbol **“G”** indicates the arrangement of the lubrication holes, the standard tapping is: M 10 x 1.00 except for the light series and the solid section light series: M 8 x 1.00.

The raceway mean diameter of the bearing is given for information only.

The gap diameters cannot be used as spigots, except when their dimensions are toleranced on the drawing.

This document is not contractual, the latest issue of the drawing is available upon request.
CAPACITY CURVES

A rating graph illustrating the various load capacities is associated with each bearing reference. This "capacity curve" shows the same reference number as the bearing. The steel specification which is used is indicated by the code letter.

- **This dotted line symbolizes the maximum dynamic capacity of the bearing.** The functional point revised with the application and utilization factors must never exceed this limit in normal operation.

- **This solid line represents a utilization limit curve for a minimum application factor of 1.25.**

- **This dot-dash line indicates the bolting limit capacity for a supported load using bolts grade 10.9.**

- **The functional point revised by utilization factors must never exceed this limit in normal operation.**

- **The title block indicates the main characteristics of the gear: m, z, x_m, and the ultimate resistance to fatigue bending stress according to the gear ring material and heat treatment indicated by a material code letter.**

---

**CURVES OF BEARING 06 1116 00**

<table>
<thead>
<tr>
<th>Gear</th>
<th>m</th>
<th>z</th>
<th>x_m</th>
<th>Z0</th>
<th>Z1</th>
</tr>
</thead>
<tbody>
<tr>
<td>m</td>
<td>10</td>
<td></td>
<td></td>
<td>116.45 kN</td>
<td>155.30 kN</td>
</tr>
<tr>
<td>z</td>
<td>125</td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>x_m</td>
<td>+ 10.50</td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>Z0</td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>Z1</td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
</tbody>
</table>
BEARING REFERENCES

BEARINGS WITH EXTERNAL GEAR - Classification according to external diameter

<table>
<thead>
<tr>
<th>External diameter</th>
<th>Internal diameter</th>
<th>Height</th>
<th>Weight</th>
<th>Module</th>
<th>Number of teeth</th>
<th>Bearing reference</th>
<th>Page</th>
<th>Quality</th>
</tr>
</thead>
<tbody>
<tr>
<td>244</td>
<td>125</td>
<td>25</td>
<td>5</td>
<td>2</td>
<td>120</td>
<td>01 0181 02</td>
<td>75</td>
<td>STD</td>
</tr>
<tr>
<td>318</td>
<td>169</td>
<td>45</td>
<td>14</td>
<td>3</td>
<td>104</td>
<td>01 0235 00</td>
<td>75</td>
<td>STD</td>
</tr>
<tr>
<td>379</td>
<td>210</td>
<td>45</td>
<td>20</td>
<td>4</td>
<td>92</td>
<td>01 0289 06</td>
<td>75</td>
<td>STD</td>
</tr>
<tr>
<td>403,5</td>
<td>234</td>
<td>55</td>
<td>25</td>
<td>4,5</td>
<td>88</td>
<td>06 0307 00</td>
<td>59</td>
<td>STD</td>
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<td>440</td>
<td>265</td>
<td>50</td>
<td>28</td>
<td>4,5</td>
<td>95</td>
<td>01 0342 00</td>
<td>76</td>
<td>STD</td>
</tr>
<tr>
<td>529</td>
<td>323</td>
<td>54</td>
<td>45</td>
<td>5</td>
<td>103</td>
<td>01 0422 01</td>
<td>76</td>
<td>STD</td>
</tr>
<tr>
<td>535</td>
<td>305</td>
<td>75</td>
<td>61</td>
<td>8</td>
<td>65</td>
<td>06 0400 00</td>
<td>59</td>
<td>STD</td>
</tr>
<tr>
<td>589</td>
<td>383</td>
<td>75</td>
<td>62</td>
<td>5</td>
<td>116</td>
<td>06 0475 22</td>
<td>59</td>
<td>STD</td>
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<td>390</td>
<td>85</td>
<td>98</td>
<td>8</td>
<td>80</td>
<td>06 0508 00</td>
<td>60</td>
<td>STD</td>
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<td>689</td>
<td>455</td>
<td>74</td>
<td>89</td>
<td>6</td>
<td>112</td>
<td>01 0555 01</td>
<td>76</td>
<td>STD</td>
</tr>
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<td>700</td>
<td>479</td>
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<td>85</td>
<td>6</td>
<td>114</td>
<td>06 0574 09</td>
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<td>STD</td>
</tr>
<tr>
<td>774</td>
<td>516</td>
<td>82</td>
<td>118</td>
<td>8</td>
<td>94</td>
<td>01 0626 00</td>
<td>77</td>
<td>STD</td>
</tr>
<tr>
<td>816</td>
<td>573</td>
<td>90</td>
<td>129</td>
<td>6</td>
<td>132</td>
<td>06 0675 00</td>
<td>60</td>
<td>STD</td>
</tr>
<tr>
<td>863</td>
<td>679</td>
<td>82</td>
<td>102</td>
<td>6</td>
<td>142</td>
<td>01 0765 01</td>
<td>77</td>
<td>STD</td>
</tr>
<tr>
<td>886</td>
<td>610</td>
<td>85</td>
<td>155</td>
<td>8</td>
<td>108</td>
<td>06 0734 00</td>
<td>61</td>
<td>STD</td>
</tr>
<tr>
<td>979</td>
<td>717</td>
<td>100</td>
<td>178</td>
<td>10</td>
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The following pages show you a selection of our range of standard bearings.
These bearings are manufactured on a regular schedule.
Consult our Sales Department about their availability.
LIGHT SERIES BEARINGS

DRAWING AND CURVES - EXTERNAL GEAR

FAMILIES : 21 - 24 - 27

The mounting holes are equispaced.
Ne : number of holes in the outer ring.
Ni : number of holes in the inner ring.

Tapping depth in the outer ring : 20 mm minimum.

G = 4 equispaced grease nipples (M8 X 1.00) mounted on the inner ring.

Material : Improved XC 45, code Z.

These bearings are also available without drilling. The references are identical except for the last two digits :
00 : without bolt holes,
01 : standard drilling.

The values on "Light Series" capacity curves can be read directly; you do not have to add any factors.
# LIGHT SERIES BEARINGS

## REFERENCES - EXTERNAL GEAR

### FAMILIES : 21 - 24 - 27

## MEASUREMENTS IN MILLIMETERS

<table>
<thead>
<tr>
<th>REFERENCES</th>
<th>WEIGHT Kg</th>
<th>DIMENSIONS</th>
<th>FASTENING</th>
<th>GEARS z</th>
<th>Teeth resistance kN</th>
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<tbody>
<tr>
<td>21 0411 01</td>
<td>32</td>
<td>505 304</td>
<td>368</td>
<td>455 10</td>
<td>332 12 5 99 495</td>
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<tr>
<td>21 0541 01</td>
<td>44</td>
<td>640 434</td>
<td>498</td>
<td>585 14</td>
<td>462 14 6 105 630</td>
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<tr>
<td>21 0641 01</td>
<td>52</td>
<td>742 534</td>
<td>598</td>
<td>685 16</td>
<td>562 16 6 122 732</td>
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<td>840 634</td>
<td>698</td>
<td>785 18</td>
<td>662 16 6 138 828</td>
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<td>21 0841 01</td>
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<td>950 734</td>
<td>798</td>
<td>885 18</td>
<td>762 18 8 117 936</td>
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<td>21 1091 01</td>
<td>86</td>
<td>1198 984</td>
<td>1048</td>
<td>1135 22</td>
<td>1012 20 8 148 1184</td>
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</table>

Families 21 : class I  24 : class III  27 : class V have all the same geometry

The characteristics of these classes are tabulated on the table page 106.
Please consult us, for the class suitable for your application.
LIGHT SERIES BEARINGS

DRAWING AND CURVES - INTERNAL GEAR

FAMILIES : 22 - 25 - 28

The mounting holes are equispaced.
Ne : number of holes in the outer ring.
Ni : number of holes in the inner ring.
Tapping depth in the inner ring:
20 mm minimum.

G = 4 equispaced grease nipples
(M8 X 1.00) mounted on the outer ring.

Material : Improved XC 45, code Z.

These bearings are also available without
drilling. The references are identical except
for the last two digits:
00 : without bolt holes,
01 : standard drilling.

The values on "Light Series" capacity curves
can be read directly; you do not have
to add any factors.
LIGHT SERIES BEARINGS

REFERENCES - INTERNAL GEAR

FAMILIES : 22 - 25 - 28

MEASUREMENTS IN MILLIMETERS

<table>
<thead>
<tr>
<th>REFERENCES</th>
<th>WEIGHT Kg</th>
<th>DIMENSIONS De</th>
<th>Di</th>
<th>Be</th>
<th>Fe</th>
<th>Ne</th>
<th>Fi</th>
<th>Ni</th>
<th>m</th>
<th>z</th>
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<td>620</td>
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<td>505</td>
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<td>6</td>
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<td>22 0641 01</td>
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<td>546</td>
<td>684</td>
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<td>93</td>
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<td>40,3</td>
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<td>649</td>
<td>784</td>
<td>820</td>
<td>12</td>
<td>705</td>
<td>20</td>
<td>6</td>
<td>110</td>
<td>660</td>
<td>40,3</td>
</tr>
<tr>
<td>22 0841 01</td>
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<td>22 0941 01</td>
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<td>24</td>
<td>8</td>
<td>125</td>
<td>1000</td>
<td>54,3</td>
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</tbody>
</table>

Families 22 : class I have all the same geometry.
Families 25 : class III 28 : class V have all the same geometry.

The characteristics of these classes are tabulated on the table page 106.
Please consult us, for the class suitable for your application.
LIGHT SERIES BEARINGS

DRAWING AND CURVES - WITHOUT GEAR

FAMILIES : 23 - 26 - 29

The mounting holes are equispaced.
Ne : number of holes in the outer ring.
Ni : number of holes in the inner ring.

G = 4 equispaced grease nipples (M8 X 1.00) mounted on each ring.

Material : Improved XC 45, code Z.

These bearings are also available without drilling. The references are identical except for the last two digits:
00 : without bolt holes,
01 : standard drilling.

The values on “Light Series” capacity curves can be read directly; you do not have to add any factors.
# LIGHT SERIES BEARINGS

## REFERENCES - WITHOUT GEAR

**FAMILIES : 23 - 26 - 29**

<table>
<thead>
<tr>
<th>REFERENCES</th>
<th>WEIGHT Kg</th>
<th>DIMENSIONS</th>
<th>FASTENING</th>
</tr>
</thead>
<tbody>
<tr>
<td></td>
<td></td>
<td>De  Di  Be  Bi  Fe  Ne  Fi  Ni</td>
<td></td>
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<tr>
<td>23 0411 01</td>
<td>23</td>
<td>518 304 454 368 490 8 332 12</td>
<td></td>
</tr>
<tr>
<td>23 0541 01</td>
<td>34</td>
<td>648 434 584 498 620 10 462 14</td>
<td></td>
</tr>
<tr>
<td>23 0641 01</td>
<td>40</td>
<td>748 534 684 598 720 12 562 16</td>
<td></td>
</tr>
<tr>
<td>23 0741 01</td>
<td>46</td>
<td>848 634 784 698 820 12 662 16</td>
<td></td>
</tr>
<tr>
<td>23 0841 01</td>
<td>52</td>
<td>948 734 884 798 920 14 762 18</td>
<td></td>
</tr>
<tr>
<td>23 0941 01</td>
<td>58</td>
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<td></td>
</tr>
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<td>23 1091 01</td>
<td>67</td>
<td>1198 984 1134 1048 1170 16 1012 20</td>
<td></td>
</tr>
</tbody>
</table>

Families 23 : class I  26 : class III  29 : class V have all the same geometry.

The characteristics of these classes are tabulated on the table page 106.
Please consult us, for the class suitable for your application.
LIGHT SERIES SOLID SECTIONS BEARINGS

DRAWING AND CURVES - EXTERNAL GEAR

FAMILIES : 31 - 34 - 37

The mounting holes are equispaced.
Ne : number of holes in the outer ring.
Ni : number of holes in the inner ring.
Tapping depth in the outer ring :
20 mm minimum.

G = 4 equispaced grease nipples
(M8 X 1.00) mounted on the inner ring.

Material : Improved XC 45, code Z.

Pilot Ce : height 3 mm, quality H9.
Pilot Ci : height 10 mm, quality H9.

These bearings are also available with
through holes. The references are identical
except for the last two digits :
01 : through holes inner ring,
11 : through holes outer ring.

The values on “Light Series” capacity curves
can be read directly ; you do not have to
add any factors.
# LIGHT SERIES SOLID SECTIONS BEARINGS

## REFERENCES - EXTERNAL GEAR

**FAMILIES : 31 - 34 - 37**

### MEASUREMENTS IN MILLIMETERS

| REFERENCES | WEIGHT Kg | De | Di | Ce | Cl | Fe | Ne | Fi | Ni | m | z | D | Teeth Resistance kN |
|------------|-----------|----|----|----|----|----|----|----|----|----|   |   |               |
| 31 0411 01 | 32        | 505| 342| 412,2 | 344| 455| 20 | 368| 24 | 5  | 99 | 495 | 33,8          |
| 31 0541 01 | 46        | 640| 472| 542,5 | 474| 585| 28 | 498| 32 | 6  | 105| 630 | 41            |
| 31 0641 01 | 55        | 742| 572| 642   | 574| 685| 32 | 598| 36 | 6  | 122| 732 | 41            |
| 31 0741 01 | 62        | 840| 672| 743   | 674| 785| 36 | 698| 40 | 6  | 138| 828 | 41            |
| 31 0841 01 | 71        | 950| 772| 842   | 774| 885| 36 | 798| 40 | 8  | 117| 936 | 55            |
| 31 0941 01 | 78        | 1046| 872| 943   | 874| 985| 40 | 898| 44 | 8  | 129| 1032 | 55           |
| 31 1091 01 | 91        | 1198| 1022|1094  |1024|1135| 44 |1048| 48 | 8  | 148| 1184 | 55           |

**Families 31 : class I  34 : class III  37 : class V** have all the same geometry.

The characteristics of these classes are tabulated on the table page 106.

Please consult us, for the class suitable for your application.
LIGHT SERIES SOLID SECTIONS BEARINGS

DRAWING AND CURVES - INTERNAL GEAR

FAMILIES: 32 - 35 - 38

The mounting holes are equispaced.

Ne: number of holes in the outer ring.
Ni: number of holes in the inner ring.

Tapping depth in the inner ring: 20 mm minimum.

G = 4 equispaced grease nipples (M8 X 1.00) mounted on the outer ring.

Material: Improved XC 45, code Z.

Spigot Ce: height 10 mm, quality f9.
Spigot Ci: height 3 mm, quality f9.

The values on “Light Series” capacity curves can be read directly; you do not have to add any factors.
# LIGHT SERIES SOLID SECTIONS BEARINGS

## REFERENCES - INTERNAL GEAR

### FAMILIES: 32 - 35 - 38

## MEASUREMENTS IN MILLIMETERS

| REFERENCES | WEIGHT Kg | DE | DI | CE | CI | FE | NE | FI | NI | M | Z | D | Teeth Resistance kN |
|------------|-----------|----|----|----|----|----|----|----|----|----|---|---|---|---------------------|
| 32 0411 01 | 31        | 486| 325| 484| 409,5| 460| 24 | 375| 24 | 5 | 67 | 335| 32,4               |
| 32 0541 01 | 43        | 616| 445| 614| 539,5| 590| 32 | 505| 32 | 6 | 76 | 456| 40,3               |
| 32 0641 01 | 51        | 716| 546| 714| 639| 690| 36 | 605| 36 | 6 | 93 | 558| 40,3               |
| 32 0741 01 | 58        | 816| 649| 814| 739,7| 790| 40 | 705| 40 | 6 | 110| 660| 40,3               |
| 32 0841 01 | 70        | 916| 736| 914| 840| 890| 40 | 805| 40 | 8 | 94 | 752| 54,3               |
| 32 0941 01 | 76        | 1016| 840| 1014| 939| 990| 44 | 905| 44 | 8 | 107| 856| 54,3               |
| 32 1091 01 | 92        | 1166| 986| 1164| 1089| 1140| 48 | 1055| 48 | 8 | 125| 1000| 54,3             |

Families 32: class I | 35: class III | 38: class V have all the same geometry.

The characteristics of these classes are tabulated on the table page 106.
Please consult us, for the class suitable for your application.
LIGHT SERIES SOLID SECTIONS BEARINGS

DRAWING AND CURVES - WITHOUT GEAR

FAMILIES : 33 - 36 - 39

The mounting holes are equispaced.
Ne : number of holes in the outer ring.
Ni : number of holes in the inner ring.

G = 4 equispaced grease nipples
(M8 X 1.00) mounted on the outer ring.

Material : Improved XC 45, code Z.

Spigot Ce : height 10 mm, quality f9.
Pilot Ci : height 10 mm, quality H9.

The values on "Light Series" capacity curves can be read directly; you do not have to add any factors.
LIGHT SERIES SOLID SECTIONS BEARINGS

REFERENCES - WITHOUT GEAR

FAMILIES : 33 - 36 - 39

<table>
<thead>
<tr>
<th>REFERENCES</th>
<th>WEIGHT Kg</th>
<th>MEASUREMENTS IN MILLIMETERS</th>
</tr>
</thead>
<tbody>
<tr>
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<td>Di</td>
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<tr>
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<tr>
<td>33 0541 01</td>
<td>38</td>
<td>616</td>
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<tr>
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<td>916</td>
</tr>
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<td>33 0941 01</td>
<td>67</td>
<td>1016</td>
</tr>
<tr>
<td>33 1091 01</td>
<td>78</td>
<td>1166</td>
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</tbody>
</table>

Families 33 : class I  36 : class III  39 : class V have all the same geometry.

The characteristics of these classes are tabulated on the table page 106.

Please consult us, for the class suitable for your application.
## LIGHT SERIES AND SOLID SECTIONS

### TABLE OF DEFLECTION UNDER MASTER LOAD

<table>
<thead>
<tr>
<th>RACEWAY MEAN DIAM. (mm)</th>
<th>MASTER LOAD (N)</th>
<th>MAXIMUM DEFLECTION (average 3 points)</th>
<th>MAXIMUM MEASUREMENT IN 1 POINT</th>
</tr>
</thead>
<tbody>
<tr>
<td></td>
<td></td>
<td>Class I</td>
<td>Class III</td>
</tr>
<tr>
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<td>0,30</td>
<td>0,15</td>
</tr>
<tr>
<td>641</td>
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<tr>
<td>1091</td>
<td>400</td>
<td>0,50</td>
<td>0,25</td>
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</tbody>
</table>
APPENDIX

- COMMISSIONING DOCUMENTS
- TECHNICAL DATA SHEET FOR THE SELECTION OF A SLEWING RING
- WARRANTY - AFTER-SALES SERVICE
You have taken receipt of a "ROLLIX" slewing ring and we congratulate you for your choice.

The "slewing ring" product is a high technology product selected for precise parameters. It was designed and manufactured to the ISO 9000 standard and supplied based on the informations given in the questionnaire IT ETR 911. It will give you trouble free service if the operating conditions are in line with "ROLLIX" recommendations.

You are a designer, an installer or a retailer, it is your responsibility to check that your or your user’s need have been correctly identified in our questionnaire IT ETR 911, that the installation requirements and our product maintenance conditions have to be fully respected.

We suggest you study the conditions in this catalogue and our manual and instruction booklet IT ETR 941 so that you are certain of your responsibilities. These documents should be supplied to the user if the situation requires it.

In order to help you, we enclose a checklist which although not comprehensive, covers the minimum essential checks which have to be carried out in all cases.

In addition we suggest you make yourself aware of our guarantee conditions and their limitations.

**CHOICE OF THE SLEWING RING**

To choose a suitable slewing ring, did you take into consideration:

- the nominal loads
- the overloads due to:
  - wind
  - snow
  - operating conditions
- the loads due to dynamic effects
- peak loads
- test overloads

This for:
- the raceway
- the gear
- the bolting
- the maximum utilization speed

On receipt, be sure that the slewing ring you receive is the one you have ordered (identification plate).

**Storage conditions,**

- during the storage time, has the product been protected against external contamination? 
- if storage has exceeded 18 months, has the slewing ring been regreased before installation?
### COMMISSIONING DOCUMENTS

Before installation, be sure that the product has not been subjected to shock loads resulting in an alteration of its characteristics (irreversible deformation),

- seals are in place
- seals are not damaged
- your handling equipment are suitable for the product:
  - quality
  - size
  - weight

During installation, check:

- the flatness of the supporting base
- the dimensions of the structures
- the position of the filler plug
- the position of the hardening junction in relation to the axis of maximum moments.

During fastening, check:

- the grade of screws and bolts
- the number of bolts used
- the specified tightening torque
- that the tightening torque is achieved using properly lubricated threads
- that split, fan-shaped or elastic washers are not used
- that hardened flat washers are used if required

Installation of the pinion gear (if necessary), check:

- the pinion corresponds to the gear of the slewing ring
- there is no risk of interference
- the minimum backlash is maintained
- the measurement is determined at the point of:
  - maximum offset of the external gear
  - maximum offset of the internal gear
  - the slewing ring / pinion centre distance cannot vary
  - the contact area of the pinion on the slewing ring gear face is uniform

Before installation, be sure:

- the maximum slewing ring speed will not exceed the calculated value
- no objects will foul the slewing ring rotation
- no objects will damage the seals
- the slewing ring and its pinion do not cause any danger
  - to material
  - to people
- that protection complies with the recommendations of 89/392 EEC
COMMISSIONING DOCUMENTS

- Check
  - the deflection under the maximum design load and record the results
  - the rotation is without abnormal noise or tight spots

After the installation,
- be sure,
- no deformation can affect the functioning of the assembly
- no deformation can decrease the life of the product

In operation,
- be sure that the user:
- knows the lubrication specifications and in particular:
  - type of grease
  - frequency of lubrication
  - the method of regreasing
  - the rolling elements
  - the gear
  - will observe the condition of seals
- knows the frequency
  - of checking bolt tightening
  - of bolt replacement
- knows the maximum permissible clearance in the product before dismounting
- or the designed maximum clearance allowable

- For your information:
  A manuel and instruction booklet IT ETR 941 is available upon request at our Commercial Department. We advise you to include our maintenance specifications in those of the machine or failing that, to transmit them to the user.
### DESIGN DATA SHEET

<table>
<thead>
<tr>
<th>1.1 COMPANY :</th>
<th></th>
<th>Tel. :</th>
<th></th>
</tr>
</thead>
<tbody>
<tr>
<td>1.2 Address :</td>
<td></td>
<td>Fax :</td>
<td></td>
</tr>
<tr>
<td>1.3 Person in charge of the project :</td>
<td></td>
<td>Fonction :</td>
<td></td>
</tr>
</tbody>
</table>

#### DESCRIPTION OF THE APPLICATION + sketch (to be attached with loads applied)

| 2.1 Machine / project reference : |  | New project : | Yes ☐ No ☐ |
| 2.2 Slowing ring position : Horiz. ☐ Vert. ☐ Other ☐ |  | Utilisation mode : Continu. ☐ Intermittent ☐ Other ☐ |

#### LOADS ON THE BEARING (including structural loads):

| 3.1 | Applied ☐ | Suspended ☐ |

##### Static

<table>
<thead>
<tr>
<th>3.2</th>
<th>Nominal</th>
<th>Maximum</th>
<th>Test</th>
<th>Nominal</th>
<th>Maximum</th>
<th>Test</th>
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<tbody>
<tr>
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<td>% Time</td>
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<td>kN</td>
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<td>kNm</td>
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<td></td>
<td></td>
<td></td>
</tr>
</tbody>
</table>

#### Rotating ring:

| 3.7 | Ext ☐ | Int ☐ | Speed (RPM) |  |

##### Load factors applied:

| 3.8 | Excluded ☐ | Included ☐ | Values : |  |

#### Required life:

| 3.9 |  |  |  |  |

##### GEAR

<table>
<thead>
<tr>
<th>4.1</th>
<th>External ☐</th>
<th>Internal ☐</th>
<th>Without gear ☐</th>
<th></th>
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<tbody>
<tr>
<td>4.2</td>
<td>Geometry</td>
<td>S.R.</td>
<td>Pinion</td>
<td>Loads</td>
</tr>
<tr>
<td>4.3</td>
<td>Required Module</td>
<td>Tangentiel load (kN)</td>
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<td></td>
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<tr>
<td>4.4</td>
<td>Number of teeth required</td>
<td>Torque (kNm)</td>
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<tr>
<td>4.5</td>
<td>Addendum modif.factor</td>
<td></td>
<td></td>
<td></td>
</tr>
</tbody>
</table>

#### CRITICAL ITEMS

| 5.1 Classification ☐ | Commission ☐ | FEM ☐ | LLOYDS ☐ | API ☐ | BV ☐ | DNV ☐ | Other |  |
| 5.2 Specification relative to the application : SPECIFIC REQUIREMENTS OF THE APPLICATION |

#### Environment :

| 6.1 | Operating temperature : |  |  |
| 6.2 | Storing temperature : |  |  |

#### Vibrations, shocks loads :

| 6.3 | Deceleration : | Inert. mom./Rotation axis : | Varied : |  |

#### Acceleration :

| 6.4 |  |  |  |  |

#### Quantity :

| 7.1 | Yearly requirements : | Qty per delivery : |  |  |

#### REQUIRED DELIVERY TIME :

| 8.1 |  |  |  |  |
WARRANTY - AFTER-SALES SERVICE

OBJECT OF THE WARRANTY
The ROLLIX DEFONTAINE "SLEWING RING" product is warranted for good working within the scope of normal usage under the following criteria:

• for standard products, usage is as per the conditions of application, the technical specifications as indicated in the catalogue reference IT ETR 921 and the instruction manual IT ETR 941 applicable to the product.
• for all products, usage is as per the indications provided by the customer in answer to the questionnaire reference IT ETR 911.

WARRANTY PERIOD
The warranty for slewing ring gears is for TWO YEARS after delivery, and FIVE YEARS after delivery with regard to defects in:

• Material origin,
• Manufacture
• Factory assembly.

For geometric defects, the warranty period however remains fixed at TWO YEARS after delivery.

In all cases, the warranty period is limited to the estimated service life of the product as shown in our calculation sheets or other ROLLIX documents. This theoretical value comes from a statistical calculation and can in no way be considered as a commitment as to the result.

CONDITIONS OF APPLICATION - EXCLUSIONS
The warranty of the "SLEWING RING" product with respect to the application will not apply in the various following cases:

1. The questionnaire reference IT ETR 911 has not been completed and returned to ROLLIX.

2. Non-compliance with ROLLIX recommendations
   Handling, utilization, mounting and maintenance which do not comply with our recommendations listed on the maintenance manual of the product sold, notably:
   • Handling
     Handling equipment not suitable to the quality and weight of the parts.
   • Utilization
     Utilization not complying with the technical specifications and with the conditions of application of the product.
   • Mounting
     Incorrect dimensioning of the support structures under minimum stated limits and flatness out of tolerances.
     Incorrect positioning of the filler plug with respect to the axis of moments.

The supply of a product suitable to customer’s needs is based on his precise and complete answers to the questionnaire IT ETR 911. Application conditions which fall outside the stipulations thereon may constitute a case of exclusion of the ROLLIX warranty.
WARRANTY - AFTER-SALES SERVICE

Fastening
- Screws and nuts not complying to our recommendations as well as to the required pretensioning.
- Utilization of split, fan-shaped, elastic or non-hardened washers whatever the brand or the model.

Greasing
Non-compliance with recommended types of grease as well as regreasing frequency.

Storage
- Non renewal of the external protective substance after six month storage.
- No regreasing of the slewing ring in rotation after eighteen month storage.

Modification, dismantling or repair of the product carried out without ROLLIX previous agreement.

Damage resulting from normal wear of the material
The increase of clearance under load within the scope of the utilization limits constitutes normal wear.

The warranty is not applicable to the wearing parts such as seals.

More generally, the warranty will not be applied in case of malfunctions resulting directly or indirectly from a fault or an act of negligence of the customer.

IMPLEMENTATION OF THE WARRANTY

The warranty for the satisfactory functioning of the "SLEWING RING" product will be granted after:
- notification of malfunction by the customer in the allotted time (see page 112 : WARRANTY PERIOD)
- observation of the malfunction by ROLLIX
- examination of the respective product's technical specifications, ROLLIX previous recommendations and other conditions applicable to the warranty.

No return will be accepted without ROLLIX prior agreement, that is to say after receipt of the duly filled in appendix 1 of the procedure PG.CRX.19.00.

WARRANTY

In the case of notified malfunction observed, and after carrying out the usual examinations, ROLLIX will proceed to:
- repair of the SLEWING RING with possible replacement of the parts considered damaged,
- or arrange for the free replacement of the SLEWING RING,
- or arrange for the reimbursement of the sale value of the SLEWING RING.

In all cases, ROLLIX reserves the right to select the most suitable solution for the problem observed.

The ROLLIX warranty is strictly limited to repair, replacement or reimbursement of the part considered damaged, and therefore will not be extended to any other consequential damage likely to be linked to the malfunction : machine stoppages, damage of goods other than those of the contract, loss of income...

In the case of malfunction of a "SLEWING RING" product which is linked to an act of negligence or a fault of the customer, and notably to the non compliance of the ROLLIX recommendations, the survey costs of the damaged part, including transport and all travelling expenses for technicians to the commission site will be charged to the customer's account.

CAUTION

All data and informations contained in the present catalogue have been carefully evaluated and checked. We cannot, however, be liable for any errors or omissions. We reserve the right to make all technical changes and amendments in tests, formulae and drawings without prior notice.

All earlier editions are invalid with effect from the publication of this catalogue.